

**Calculations executed for the 3-bladed rotor of the VIRYA-10 windmill ( $\lambda_d = 6$ )  
with the pendulum safety system with a torsion spring connected to  
the generator type TGET770-H-10KW-100R for grid connection**

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It is allowed to copy this report for private use. A prototype of the VIRYA-10 windmill has not yet been built and tested. This should be done only by a professional company after making detailed drawings. Although the VIRYA-10 has been designed carefully, no responsibility is accepted by Kragten Design for the proper functioning.

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## 1 Introduction

In the Dutch report KD 713 (ref. 1) it is proposed to use a windmill with a rotor diameter of 10 m to generate the energy for mainly the heat pumps of four houses. The rotor would be connected direct drive to one of the biggest PM-generators of Hefei Top Grand with housing type TGET770. Some preliminary calculations have shown that the torque level of a rotor with 10 m diameter is too high if the hinged side vane safety system is used. So a rotor with a diameter of 9 m is designed in report KD 714 (ref. 2) for use in combination to the hinged side vane safety system. However, a rotor diameter of 10 m seems allowed if the pendulum safety system with a torsion spring is used as the maximum torque level of the rotor can be restricted much sharper with this safety system. This safety system is described in report KD 439 (ref. 3). Some specific calculations for the VIRYA-10 are given in chapter 6 of KD 439. So a new rotor with a diameter of 10 m is designed in this report KD 715. The windmill is called the VIRYA-10. The VIRYA-10 will be grid connected. A windmill with a rotor diameter of 10 m is certainly too large to be built by non professionals.

For the generator it is chosen to use the biggest axial flux PM-generator of the Chinese supplier Hefei Top Grand. This generator has a housing type TGET770 and 770 is about the diameter of the housing in mm. Five different generators with different powers at different rotational speeds are supplied for this housing size. The choice will be made in chapter 6. The axial flux generators of Hefei Top Grand type TGET have an outer rotor, so the whole generator housing is rotating. It has a 3-phase winding which is connected in star internally. The three phase wires are guided through the hollow shaft. For use in combination with an inverter, the 3-phase current has to be rectified. Rectification of the winding is described in report KD 340 (ref. 4). Selection of the right rectifier and the right inverter is out of the scope of this report.

The head is kept perpendicular to the wind by a double vane with 6 m long vane arms and with square vane blades size 1 \* 1 m at the end of each vane arm. The VIRYA-10 will have a free standing tubular tower with a height of 18 m or 24 m. At this moment it is chosen to use no hot water reservoir for storage of energy like it is proposed in KD 713. The tower is placed in the back garden at the point of intersection of the parcel boundaries of four houses.

## 2 Description of the rotor of the VIRYA-10 windmill

The 3-bladed rotor of the VIRYA-10 windmill has a diameter  $D = 10$  m and a design tip speed ratio  $\lambda_d = 6$ . The rotor has blades with a constant chord and no twist and is provided with a Gö 711 airfoil. This airfoil is described in report KD 285 (ref. 5). A blade is made out of a wooden plank with dimensions of 67 \* 400 \* 4450 mm. The airfoil is made over the whole blade length. The blade has no twist so the blade angle  $\beta$  is the same for the whole blade.

The VIRYA-5B3 has a hub plate which is made out of one sheet. If this would also be done for the VIRYA-10, a very big sheet would be needed and a lot of material would be wasted. So for the VIRYA-10 it is assumed that three strips are welded together under an angle of 120°. The strips are made of mild steel strip with a thickness of 15 mm, a width of 300 mm and a length of 1000 mm. Each strip is twisted 7.4° right hand in between  $r = 350$  mm and  $r = 550$  mm to give the blade the correct blade angle. The welded hub assembly is galvanised. The overlap in between a blade and a strip is 450 mm which results in a free blade length of 4 m. The blades are connected to the hub assembly by three bolts M20 and six nuts M20. A cambered sheet size 450 \* 80 \* 4 mm is placed under the bolt heads to prevent damage of the wood when the bolts are tightened. The hub assembly is bolted to the generator housing by means of twelve bolts M12. The rotor is balanced by adding balance weights under the connecting bolts. A sketch of the VIRYA-10 rotor is given in figure 1.

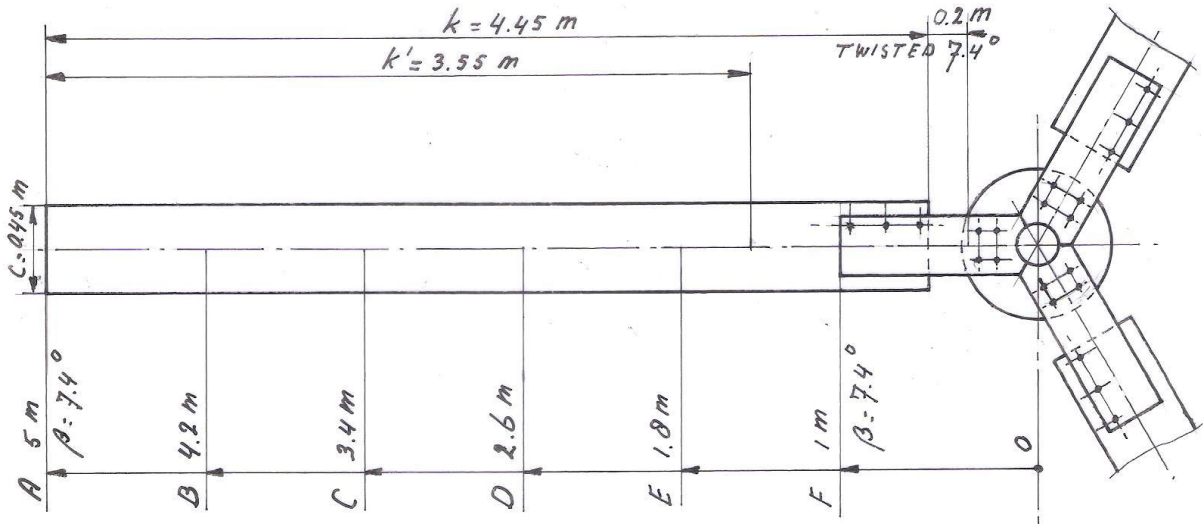


fig. 1 Sketch VIRYA-10 rotor

### 3 Calculation of the rotor geometry

The rotor geometry is determined using the method and the formulas as given in report KD 35 (ref. 6). This report (KD 715) has its own formula numbering. Substitution of  $\lambda_d = 6$  and  $R = 5$  m in formula (5.1) of KD 35 gives:

$$\lambda_{rd} = 1.2 * r \quad (-) \quad (1)$$

Formula's (5.2) and (5.3) of KD 35 stay the same so:

$$\beta = \phi - \alpha \quad (^\circ) \quad (2)$$

$$\phi = 2/3 \arctan 1 / \lambda_{rd} \quad (^\circ) \quad (3)$$

Substitution of  $B = 3$  and  $c = 0.45$  m in formula (5.4) of KD 35 gives:

$$C_l = 18.617 r (1 - \cos\phi) \quad (-) \quad (4)$$

Substitution of  $V = 5$  m/s and  $c = 0.45$  m in formula (5.5) of KD 35 gives:

$$Re_r = 1.501 * 10^5 * \sqrt{(\lambda_{rd}^2 + 4/9)} \quad (-) \quad (5)$$

The blade is calculated for six stations A till F which have a distance of 0.8 m of one to another. Cross section F corresponds to the end of a strip of the hub assembly. The blade has a constant chord and the calculations therefore correspond with the example as given in chapter 5.4.2 of KD 35. This means that the blade is designed with a low lift coefficient at the tip and with a high lift coefficient at the root.

First the theoretical values are determined for  $C_l$ ,  $\alpha$  and  $\beta$  and next  $\beta$  is linearized such that the twist is constant and that the linearized values for the outer part of the blade correspond as good as possible with the theoretical values. The result of the calculations is given in table 1.

The aerodynamic characteristics of the Gö 711 airfoil are given in report KD 285 (ref. 5). This airfoil is flat over 97.5 % of the chord and is therefore easy to manufacture. A disadvantage of this airfoil is that it has been measured only for a rather high Reynolds value of  $4 * 10^5$ . But as the VIRYA-10 has a rather large chord, this is no problem. The Reynolds values for the stations are calculated for a wind speed of 5 m/s because this is a reasonable wind speed for a windmill which is used in areas with moderate wind speeds.

station	r (m)	$\lambda_{rd}$ (-)	$\phi$ (°)	c (m)	$C_{lth}$ (-)	$C_{lin}$ (-)	$Re_r * 10^{-5}$ V = 5 m/s	$Re * 10^{-5}$ Gö 711	$\alpha_{th}$ (°)	$\alpha_{lin}$ (°)	$\beta_{th}$ (°)	$\beta_{lin}$ (°)	$C_d/C_{lin}$ (-)
A	5	6	6.3	0.45	0.56	0.57	9.06	4	-1.3	-1.1	7.6	7.4	0.025
B	4.2	5.04	7.5	0.45	0.67	0.67	7.63	4	0.1	0.1	7.4	7.4	0.020
C	3.4	4.08	9.2	0.45	0.81	0.81	6.21	4	1.8	1.8	7.4	7.4	0.016
D	2.6	3.12	11.8	0.45	1.03	1.02	4.79	4	4.5	4.4	7.3	7.4	0.015
E	1.8	2.16	16.6	0.45	1.39	1.38	3.39	4	9.3	9.2	7.3	7.4	0.023
F	1	1.2	26.5	0.45	1.96	1.30	2.06	4	-	19.1	-	7.4	0.18

table 1 Calculation of the blade geometry of the VIRYA-10 rotor

No value for  $\alpha_{th}$  and therefore for  $\beta_{th}$  is found for station F because the required  $C_l$  value can't be generated. The variation of the theoretical blade angle  $\beta_{th}$  is only little for the stations A up to E and varies in between  $7.6^\circ$  and  $7.3^\circ$ . Therefore it is allowed to take a constant value of  $\beta_{lin} = 7.4^\circ$  for the whole blade.

#### 4 Determination of the $C_p$ - $\lambda$ and the $C_q$ - $\lambda$ curves

The determination of the  $C_p$ - $\lambda$  and  $C_q$ - $\lambda$  curves is given in chapter 6 of KD 35. The average  $C_d/C_l$  ratio for the most important outer part of the blade is about 0.02. Figure 4.7 of KD 35 (for  $B = 3$ ) and  $\lambda_{opt} = 6.5$  and  $C_d/C_l = 0.02$  gives  $C_{p th} = 0.48$ .

The blade is stalling at station F. For the calculation of the maximum  $C_p$  therefore not the whole blade length  $k = 4.45$  m is taken into account but only the part up to 0.45 m outside station F. This gives an effective blade length  $k' = 3.55$  m.

Substitution of  $C_{p th} = 0.48$ ,  $R = 5$  m and effective blade length  $k' = 3.55$  m in formula 6.3 of KD 35 gives  $C_{p max} = 0.44$ .  $C_{q opt} = C_{p max} / \lambda_{opt} = 0.44 / 6 = 0.0733$ . Substitution of  $\lambda_{opt} = \lambda_d = 6$  in formula 6.4 of KD 35 gives  $\lambda_{unl} = 9.6$ . The starting torque coefficient is calculated with formula 6.12 of KD 35 which is given by:

$$C_{q start} = 0.75 * B * (R - \frac{1}{2}k) * C_l * c * k / \pi R^3 \quad (-) \quad (6)$$

The blade angle is  $7.4^\circ$  for the whole blade. For a non rotating rotor, the angle of attack  $\alpha$  is therefore  $90^\circ - 7^\circ = 82.6^\circ$ . The aerodynamic characteristics for the Gö 711 aren't given for large angles of  $\alpha$  in KD 285. However, it is assumed that the estimated  $C_l$ - $\alpha$  curve of the Gö 623 airfoil can be used for large values of  $\alpha$  which is given as figure 5.10 of KD 35 (ref. 6). For  $\alpha = 82.6^\circ$  it can be read that  $C_l = 0.26$ . The whole blade is stalling during starting and the part of the blade behind the hub assembly isn't effective. Therefore, for  $k$  now the free blade length  $k = 4$  m is taken.

Substitution of  $B = 3$ ,  $R = 5$  m,  $k = 4$  m,  $C_l = 0.26$  and  $c = 0.45$  m in formula 6 gives that  $C_{q start} = 0.0080$ . For the ratio between the starting torque and the optimum torque we find that it is  $0.0080 / 0.0733 = 0.109$ . This is acceptable for a rotor with  $\lambda_d = 6$ .

The starting wind speed  $V_{start}$  of the rotor is calculated with formula 8.6 of KD 35 which is given by:

$$V_{\text{start}} = \sqrt{\left( \frac{Q_s}{C_{q \text{ start}} * \frac{1}{2} \rho * \pi R^3} \right)} \quad (\text{m/s}) \quad (7)$$

In the specification of the generator it is mentioned that the starting torque is smaller than 0.5 Nm. This is very low and I doubt if this is correct if the generator has a seal on the shaft. The generator can be used without a seal for a vertical axis wind turbine but for a horizontal axis wind turbine, a seal is certainly necessary to prevent that water enters the bearings. Assume that the sticking torque with a seal is 4 Nm.

If the generator has no seal on the shaft and no outside chamber for mounting of an oil seal, it might be possible to use a V-ring of manufacture Forsheda type V-100A. This ring is clamped around the shaft and has an elastic lip which is pressed against the collar at the shaft side. But this is only possible if the collar surface is machined flat with a low roughness. Some grease has to be put on the lip at mounting.

Substitution of  $Q_s = 4 \text{ Nm}$ ,  $C_{q \text{ start}} = 0.0080$ ,  $\rho = 1.2 \text{ kg/m}^3$  and  $R = 5 \text{ m}$  in formula 7 gives that  $V_{\text{start}} = 1.5 \text{ m/s}$ . This is very low for a 3-bladed rotor with a design tip speed ratio  $\lambda_d = 6$  meant to be used in regions with moderate wind speeds.

In chapter 6.4 of KD 35 it is explained how rather accurate  $C_p$ - $\lambda$  and  $C_q$ - $\lambda$  curves can be determined if only two points of the  $C_p$ - $\lambda$  curve and one point of the  $C_q$ - $\lambda$  curve are known. The first part of the  $C_q$ - $\lambda$  curve is determined according to KD 35 by drawing an S-shaped line which is horizontal for  $\lambda = 0$ . Kragten Design developed a method with which the value of  $C_q$  for low values of  $\lambda$  can be determined (see report KD 97 ref. 7). With this method, it can be determined that the  $C_q$ - $\lambda$  curve is about straight and horizontal for low values of  $\lambda$  if a Gö 623 airfoil is used. A scale model of a three bladed rotor with constant chord and blade angle and with a design tip speed ratio  $\lambda_d = 6$  has been measured in the wind tunnel already on 20-11-1980. It has been found that the maximum  $C_p$  was more than 0.4 and that the  $C_q$ - $\lambda$  curve for low values of  $\lambda$  was not horizontal but somewhat rising. This effect has been taken into account and the estimated  $C_p$ - $\lambda$  and  $C_q$ - $\lambda$  curves for the VIRYA-10 rotor are given in figure 2 and 3.

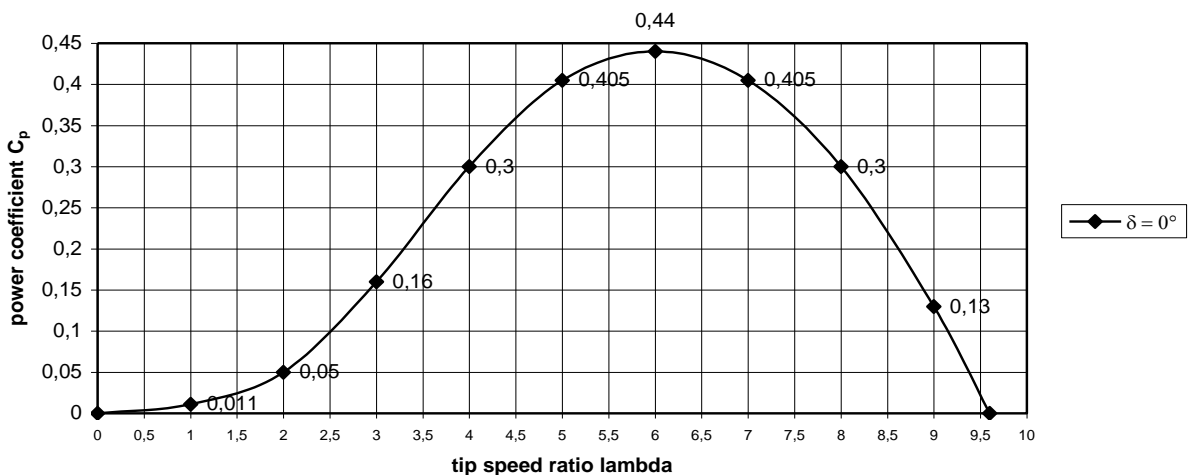


fig. 2 Estimated  $C_p$ - $\lambda$  curve for the VIRYA-10 rotor for the wind direction perpendicular to the rotor ( $\delta = 0^\circ$ )

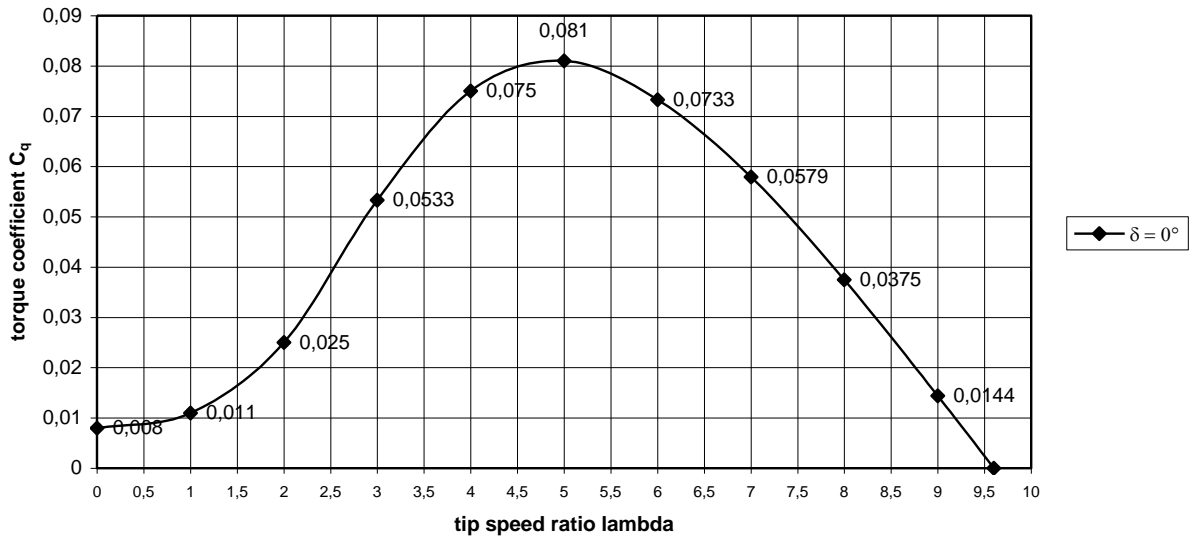


fig. 3 Estimated  $C_q$ - $\lambda$  curve for the VIRYA-10 rotor for the wind direction perpendicular to the rotor ( $\delta = 0^\circ$ )

## 5 Determination of the P-n curves and the optimum cubic line

The determination of the P-n curves of a windmill rotor is described in chapter 8 of KD 35. One needs a  $C_p$ - $\lambda$  curve of the rotor and the characteristics of the safety system together with the formulas for the power P and the rotational speed n. The  $C_p$ - $\lambda$  curve is given in figure 2.

The characteristics of the safety system are derived in chapter 3 of KD 439 for a design wind speed  $V_d = 8$  m/s. This means that the rotor starts yawing around the horizontal axis at a wind speed of 8 m/s. In figure 8, 9 and 10 of KD 439 it can be seen that the rotational speed, the thrust, the torque and the power are about maximal for a wind speed of 10 m/s. So a wind speed of 10 m/s is the rated wind speed  $V_{rated}$ . In figure 7 of KD 439 it can be seen that the yaw angle  $\delta$  is about  $30^\circ$  for  $V = 10$  m/s and that the yaw angle for  $V = 9$  m/s is about  $20^\circ$ . So the P-n curves of the rotor are determined for wind speeds up to 8 m/s for the rotor perpendicular to the wind, for  $V = 9$  m/s for a yaw angle  $\delta = 20^\circ$  and for  $V = 10$  m/s for a yaw angle  $\delta = 30^\circ$ .

The P-n curves are used to check the matching with the  $P_{mech}$ -n curve of the generator for a certain gear ratio  $i$  (the VIRYA-10 has no gearing so  $i = 1$ ). Because we are especially interested in the domain around the optimal cubic line and because the P-n curve for low values of  $\lambda$  appear to lie very close to each other, the P-n curves are not determined for low values of  $\lambda$ . The P-n curves are determined for wind the speeds 3, 4, 5, 6, 7, 8, 9 and 10 m/s. Substitution of  $R = 5$  m in formula 7.1 of KD 35 gives:

$$n = 1.9099 * \lambda * \cos\delta * V \quad (\text{rpm}) \quad (8)$$

Substitution of  $\rho = 1.2$  kg / m<sup>3</sup> and  $R = 5$  m in formula 7.10 of KD 35 gives:

$$P = 47.124 * C_p * \cos^3\delta * V^3 \quad (\text{W}) \quad (9)$$

The P-n curves are determined for  $C_p$  values belonging to  $\lambda$  is 3, 4, 5, 6, 7, 8, 9 and 9.6 (see figure 2). For a certain wind speed, for instance  $V = 3$  m/s, related values of  $C_p$  and  $\lambda$  are substituted in formula 8 and 9 and this gives the P-n curve for that wind speed. The result of the calculations is given in table 2.

The calculated values for  $n$  and  $P$  are plotted in figure 4. The optimum cubic line which can be drawn through the tops of the  $P$ - $n$  curves, is also given in figure 4.

$\lambda$	$C_p$	V = 3 m/s $\delta = 0^\circ$		V = 4 m/s $\delta = 0^\circ$		V = 5 m/s $\delta = 0^\circ$		V = 6 m/s $\delta = 0^\circ$		V = 7 m/s $\delta = 0^\circ$		V = 8 m/s $\delta = 0^\circ$		V = 9 m/s $\delta = 20^\circ$		V = 10 m/s $\delta = 30^\circ$	
		n (rpm)	P (W)	n (rpm)	P (W)	n (rpm)	P (W)	n (rpm)	P (W)	n (rpm)	P (W)	n (rpm)	P (W)	$n_s$ (rpm)	$P_s$ (W)	$n_s$ (rpm)	$P_s$ (W)
3	0.16	17.2	204	22.9	483	28.6	942	34.4	1629	40.1	2586	45.8	3860	48.5	4561	49.6	4897
4	0.3	22.9	382	30.6	905	38.2	1767	45.8	3054	53.5	4849	61.1	7238	64.6	8552	66.2	9182
5	0.405	28.6	515	38.2	1221	47.7	2386	57.3	4122	66.8	6546	76.4	9772	80.8	11545	82.7	12396
6	0.44	34.4	560	45.8	1327	57.3	2592	68.8	4479	80.2	7112	91.7	10616	96.9	12542	99.2	13467
7	0.405	40.1	515	53.5	1221	66.8	2386	80.2	4122	93.6	6546	107.0	9772	113.1	11545	115.8	12396
8	0.3	45.8	382	61.1	905	76.4	1767	91.7	3054	107.0	4849	122.2	7238	129.2	8552	132.3	9182
9	0.13	51.6	165	68.8	392	85.9	766	103.1	1323	120.3	2101	137.5	3137	145.4	3706	148.9	3979
9.6	0	55.0	0	73.3	0	91.7	0	110.0	0	128.3	0	146.7	0	155.1	0	158.8	0

table 2 Calculated values of  $n$  and  $P$  as a function of  $\lambda$  and  $V$  for the VIRYA-10 rotor

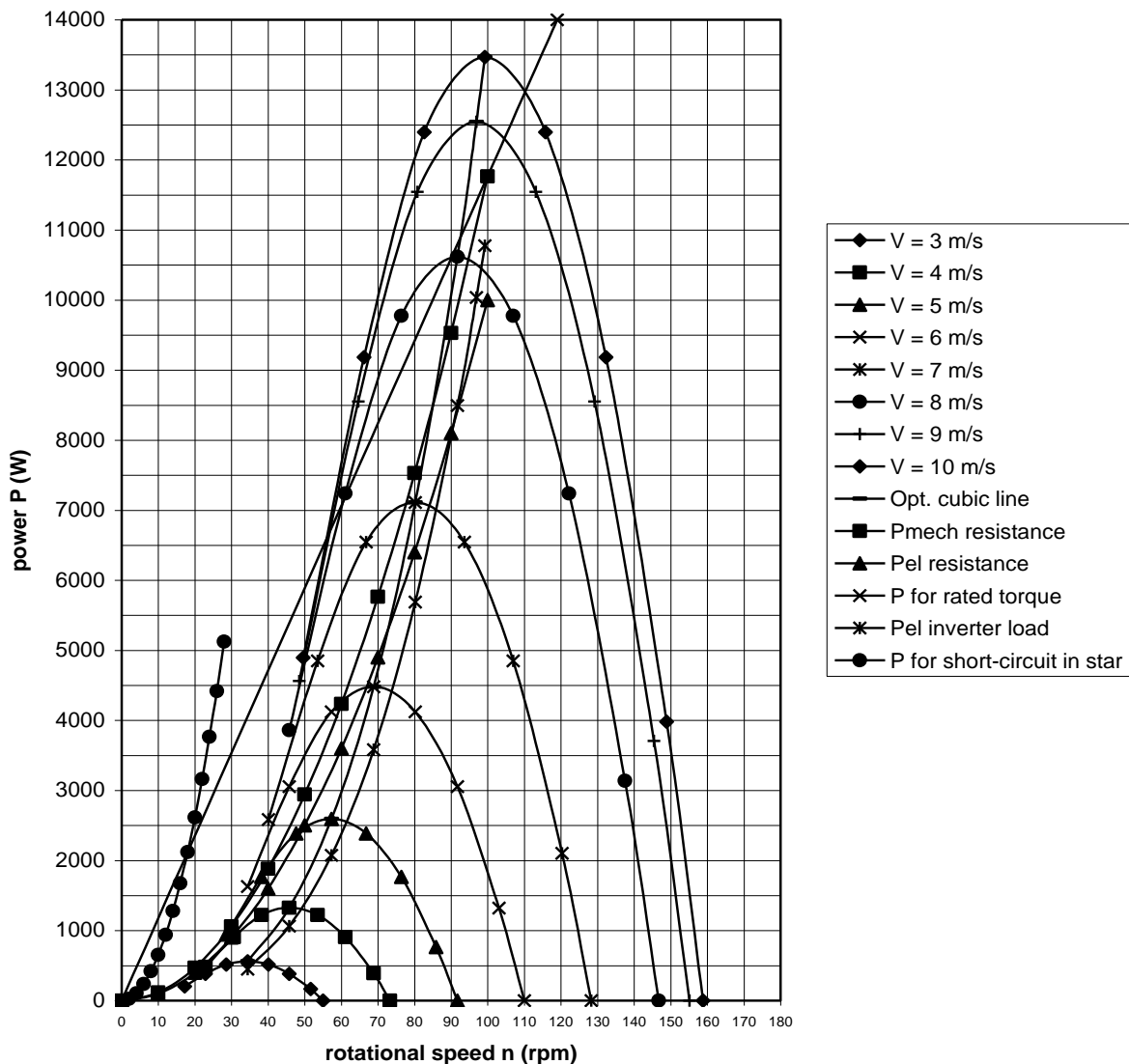


fig. 4  $P$ - $n$  curves of the VIRYA-10 rotor, optimum cubic line,  $P_{mech}$ - $n$  and  $P_{el}$ - $n$  curves for the TGET770-H-10KW-100R generator with a resistance load such that  $P_{el} = 10000$  W at  $n = 100$  rpm,  $P$ - $n$  curve for the rated torque  $Q = 1123.5$  Nm,  $P_{el}$ - $n$  curve for an inverter load,  $P$ - $n$  curve for short-circuit in star



## 6 Determination of the generator characteristics and the $P_{el}$ -V curve

An axial flux generator of Chinese manufacture has been chosen. Axial flux means that the magnetic flux which is flowing through the coils is in parallel to the generator axis. There is no iron in the coils and so the sticking torque is only determined by the friction of the bearings and the seal on the shaft. As there is no iron in the coils, there are no magnetic losses and the peak efficiency is rather high. Such generators are supplied by different Chinese suppliers like Hefei Top Grand, Xinda Green Energy, Hiestmotor and Qiangsheng Magnets. I have chosen Hefei Top Grand, website: [www.china-topgrand.com](http://www.china-topgrand.com) because they gave the clearest answers on my questions. I have bought and tested a much smaller generator type TGET165-0.15kW-500R at this company and they keep their promises. Measurements for this generator and experiments with a small wind turbine are given in report KD 595 (ref. 9).

For the VIRYA-10, I have chosen the generator with biggest available housing type TGET770 (770 refers to about the housing diameter in mm, the real diameter is 665 mm). For this housing, five different powers can be supplied depending on the rotational speed. The type number, TGET770 is followed by the power in kW and the rotational speed in rpm. The following five types are available: TGET770-7.5KW-100R, TGET770-H-10KW-100R, TGET770-10KW-150R, TGET770-45KW-300R and TGET770-60KW-450R. A data sheet about each generator can be found on the website of the supplier following the path: [www.china-topgrand.com](http://www.china-topgrand.com) – product – Permanent Magnet Generator Outer Rotor – page 4 for the 7.5 kW and 10 kW versions and page 3 for the H-10 kW, the 45 kW and 60 kW versions. In June 2021 I have tried to follow this path again but now the website is completely Chinese. I could no longer follow this path but I assume that the chosen generator is still available.

I have studied all pages and found some strange things. The first thing is that, if every generator has about the same rated torque level, it could be expected that the rated maximum power increases about linear to the rated rotational speed. But this isn't true.

It appears that two 10 kW versions are available which give the power at 100 rpm and at 150 rpm. This means that the rated torque level of the 100 rpm version must be a factor 1.5 higher. This is only possible if the 100 rpm version has stronger magnets and a stator winding which contains more copper. It will probably also be more expensive. The rated torque level of the 45 kW and the 60 kW versions must also be much higher than for the other versions.

Another strange point is that for the 7.5 kW and both 10 kW versions it is mentioned that the efficiency is larger than 85 %, which I believe. But for the 45 kW and the 60 kW version it is mentioned that the efficiency is larger than 95 % which I think is a mistake, especially because the given powers and currents are very high. Another mistake is that the dimension of the sticking torque is given as N/m in stead of Nm.

First, the TGET770-10KW-150R was chosen but the maximum torque level appeared to be too low at high wind speeds. So finally the TGET770-H-10KW-100R was chosen. So the generated electrical power is 10000 W at  $n = 100$  rpm. Next it is assumed that the efficiency is 85 % or 0.85 (-). So the required mechanical power is  $10000 / 0.85 = 11765$  W. The generator has a mass of 165 kg which seems acceptable for the VIRYA-10.

The rated loaded voltage at  $n = 100$  rpm is specified as 380 VAC. So no DC voltage is specified but the loaded DC voltage can be calculated. This generator has a 3-phase winding with an internal star point and three phase wires are coming out of the hollow generator shaft. The given voltage is the voltage in between two of the three phases and not the phase voltage  $U_f$ , which is the voltage in between the star point and one of the phases.  $U_f$  is a factor  $\sqrt{3}$  lower, so 219.39 VAC. A large 3-phase rectifier (not included) must be used to get a DC current which is needed for the inverter. Rectification of a 3-phase current is explained in report KD 340 (ref. 3). However, it might be that the rectifier is included in the inverter and in this case the three phase wires are directly connected to the inverter. To stop the rotor, a 3-phase switch has to be mounted at the tower foot. The switch must be mounted before the rectifier and as close as possible to the generator to prevent a voltage drop over the lines in between the generator and the switch.

The rated line current  $I$  is specified as  $I = 15.2$  A at  $n = 100$  rpm. So the rated power generated by one phase is  $U_f * I = 219.39 * 15.2 = 3335$  W. So the rated power generated by three phases is  $3 * 3335 = 10005$  W. This matches well with the given rated power of 10 kW. The small difference must be caused by rounding off the current.

The sticking torque of the generator is very low without an oil seal and is only caused by the friction of the bearings. It is specified that this torque is less than 0.5 Nm. An oil seal is needed if the axis is horizontal. The sticking torque will be much higher if an oil seal is mounted but it is expected that it is low enough for the VIRYA-10 rotor (see calculation of the starting wind speed in chapter 4).

The generator has a shaft with a diameter of 98 mm and this shaft will be strong enough for a horizontal axis wind turbine with a rotor diameter of 10 m. The generator housing has a collar with a diameter of 220 mm at the front side and a 220 mm hole has to be made in the hub assembly. The generator has three pairs of M12 holes with a radial pitch of 100 mm and a tangential pitch of 158 mm. The six outer threaded holes are lying at a pitch circle of 612 mm. The outer diameter of the generator is 765 mm. The outer diameter of the connection planes is 695 mm.

The generator characteristics are given in point 6 of the data sheet. The  $P_{el-n}$  and the loaded  $U-n$  curves are given. The curves show measuring points but the measuring points aren't given in a table. The  $U-n$  curve is about a straight line through the origin and the  $P_{el-n}$  curve is about a parabola. This is an indication that the load is a fixed resistance for the whole range of measurements. I have performed this kind of measurements on a radial flux PM-generator made from an asynchronous motor. These measurements are given in chapter 7 and 9 of report KD 78 (ref. 10). These measurements show that the  $P_{mech-n}$  and  $P_{el-n}$  curves are about parabolas if the resistance isn't very low, that the  $U-n$  and  $Q-n$  curves are about straight lines through the origin and that the efficiency is about constant for a certain resistance. So these curves are estimated for the given generator of Hefei Top Grand from the given rated values at  $n = 100$  rpm.

The given rated voltage is the alternating voltage  $U_{AC}$  in between two of the three phases. For an inverter, the winding must be rectified. The rectified DC voltage  $U_{DC}$  is a factor  $0.955 * \sqrt{2} = 1.3506$  higher than  $U_{AC}$  (if the voltage drop of the rectifier diodes is neglected). The unloaded or open voltage  $U_{open}$  is also not specified. For a smaller generator type TGET320-1KW-350R, it has been found in chapter 3 of report KD 705 (ref. 11) that the ratio  $U_{open} / U_{DC}$  is about  $68 / 56 = 1.2143$ . It is assumed that this ratio is also valid for the generator type TGET770-H-10KW-100R. So for the loaded DC voltage  $U_{DC}$  at  $n = 100$  rpm it is valid that  $U_{DC} = 1.3506 * 380 = 513$  VDC. For the open DC voltage  $U_{open}$  at  $n = 100$  rpm it is valid that  $U_{open} = 1.2143 * 1.3506 * 380 = 623$  VCD. The calculated values are given in the bottom line of table 3.

n (rpm)	U <sub>AC</sub> (V)	U <sub>DC</sub> (V)	U <sub>open</sub> (V)	P <sub>el</sub> (W)	η <sub>gen</sub> (-)	P <sub>mech</sub> (W)	Q (Nm)	P <sub>heat</sub> (W)
0	0	0	0	0	-	0	0	0
10	38	51.3	62.3	100	0.85	118	112.4	18
20	76	102.6	124.6	400	0.85	471	224.7	71
30	114	153.9	186.9	900	0.85	1059	337.1	159
40	152	205.2	249.2	1600	0.85	1882	449.4	282
50	190	256.5	311.5	2500	0.85	2941	561.8	441
60	228	307.8	373.8	3600	0.85	4235	674.1	635
70	266	359.1	436.1	4900	0.85	5765	786.5	865
80	304	410.4	498.4	6400	0.85	7529	898.8	1129
90	342	461.7	560.7	8100	0.85	9529	1011.2	1429
100	380	513	623	10000	0.85	11765	1123.5	1765

table 3 U<sub>AC</sub>, U<sub>DC</sub>, U<sub>open</sub>, P<sub>el</sub>, η<sub>gen</sub>, P<sub>mech</sub>, Q and P<sub>heat</sub> as a function of n

No rated torque Q is given for the generator. However, it is specified at point 5 of the data sheet that the generator efficiency η<sub>gen</sub> is at least 85 %. In figure 33 of KD 78 (ref. 10) it can be seen that the efficiency for a resistance load is about constant for every rotational speed and that it is high if the load resistance isn't low. It is easy to give the efficiency as a factor of 1 and it is assumed that η<sub>gen</sub> = 0.85 for all rotational speeds. As the generator has no iron in the coils, the heat losses P<sub>heat</sub> are only caused by the copper losses in the winding. The P<sub>mech</sub>-n, the P<sub>heat</sub>-n and the Q-n curves of the generator can be derived by the formulas:

$$P_{\text{mech}} = P_{\text{el}} / \eta_{\text{gen}} \quad (\text{W}) \quad (10)$$

$$P_{\text{heat}} = P_{\text{mech}} - P_{\text{el}} \quad (\text{W}) \quad (11)$$

$$Q = 30 P_{\text{mech}} / (\pi * n) \quad (\text{Nm}) \quad (12)$$

First the values of P<sub>mech</sub>, P<sub>heat</sub> and Q are determined for n = 100 rpm. Substitution of P<sub>el</sub> = 10000 W and η<sub>gen</sub> = 0.85 in formula 10 gives that P<sub>mech</sub> = 11765 W. Substitution of P<sub>mech</sub> = 11765 and P<sub>el</sub> = 10000 W in formula 11 gives that the heat loss P<sub>heat</sub> = 1765 W. Substitution of P<sub>mech</sub> = 11765 W and n = 100 rpm in formula 12 gives that Q = 1123.5 Nm. These values are also given in the bottom line of table 3.

The values for other rotational speeds are now calculated assuming that the U-n and Q-n curves are straight lines through the origin and that the P-n curves are parabolas. The wanted curves can now be derived from table 3. The U<sub>AC</sub>-n, the U<sub>DC</sub>-n and the U<sub>open</sub>-n curves are given in figure 5.

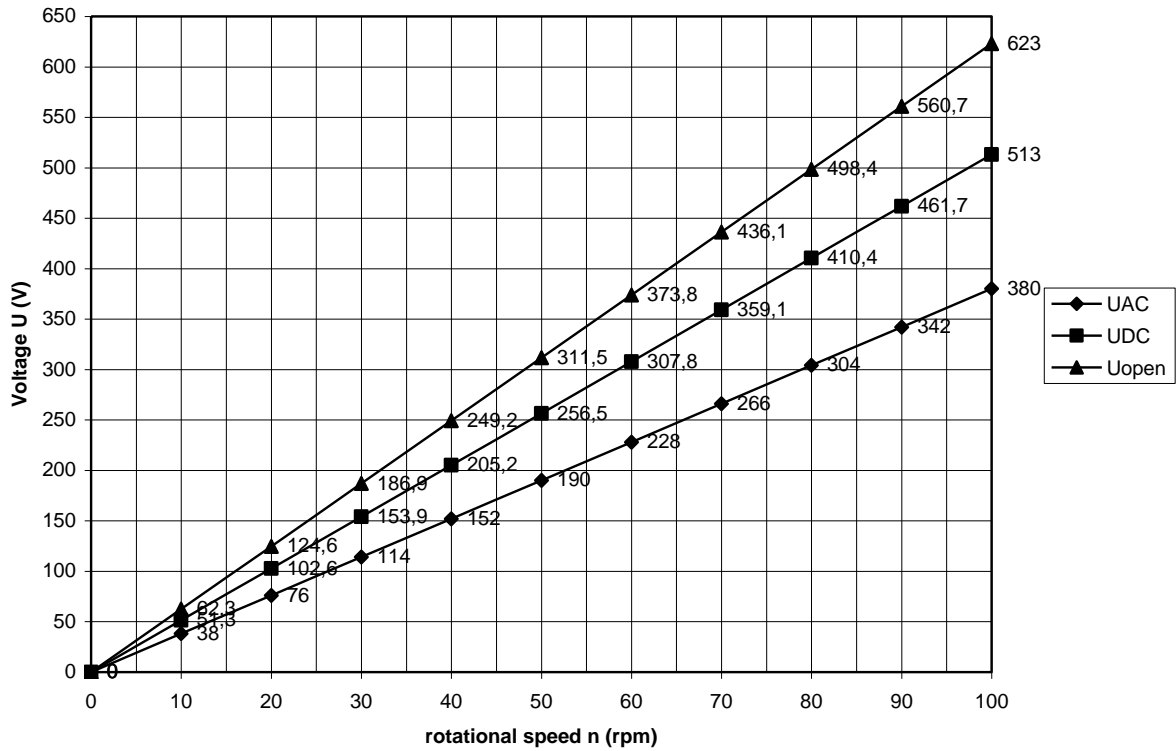


fig. 5  $U_{AC}$ ,  $U_{DC}$  and  $U_{open}$  as a function of n for a resistance load

The Q-n curve is given in figure 6. Figure 6 also contains the curve for  $U = 513$  V DC and for short-circuit in star.

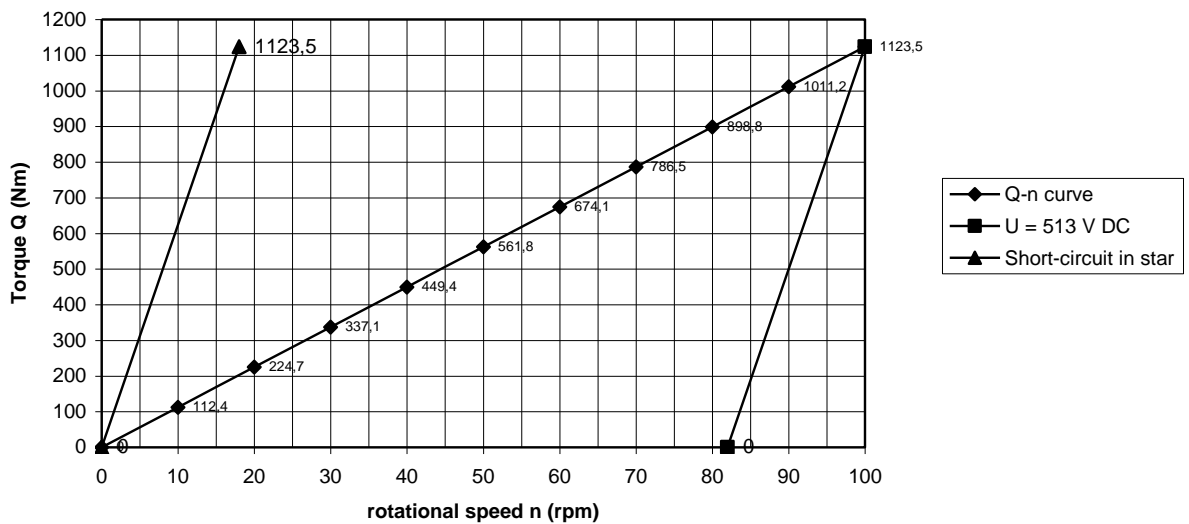


fig. 6 Loaded torque Q as a function of n for a resistance load, Q-n curves for 513 V DC and for short-circuit in star

The  $P_{mech}$ -n and  $P_{el}$ -n curves for the given resistance load are given in figure 4.

In figure 4 it can be seen that the optimum cubic line is intersecting with the  $P_{\text{mech-n}}$  line for a resistance load at a wind speed of about 7.4 m/s. So below this wind speed, the load is lower than for the given resistance load if the optimum cubic line is followed. This means that the generator efficiency is somewhat higher than for a resistance load. So it can be concluded that the VIRYA-10 with  $\lambda_d = 6$  will work well with the selected PM-generator if the inverter is programmed such that the optimum cubic line is followed and if the generator characteristics given for this generator by the manufacturer are correct.

So these figures are based on the manufactures specification for a resistance load at  $n = 100$  rpm. The load resistance  $R$  can be calculated if it is assumed that three identical resistors are connected in star to the three phase wires. The voltage over one resistor is equal to the phase voltage  $U_f = 219.39$  V. The line current  $I = 15.2$  A at  $n = 100$  rpm. So according to the law of Ohm, the resistance  $R$  is given by  $R = U / I$  or  $R = 219.39 / 15.2 = 14.4 \Omega$ .

If three resistors are used as load, the winding of one phase is used for all the time to generate power. This power varies according to a  $\sin^2\alpha$  function. The power fluctuation is given in figure 2 of report KD 340 (ref. 3). If a 3-phase winding is rectified in star, only two of the three phases are generating power at the same time. This means that in one phase, power is only generated for  $30^\circ < \alpha < 150^\circ$  and for  $210^\circ < \alpha < 330^\circ$ . This means that no power is generated for  $0^\circ < \alpha < 30^\circ$ , for  $150^\circ < \alpha < 210^\circ$  and for  $330^\circ < \alpha < 360^\circ$ . The loss of generated power because of this effect is about 7 % of the power generated for a resistance load. But this effect is neglected and so it is assumed that the generator is able to generate a DC power of 10 kW at  $n = 100$  rpm.

In the last column of table 3 it can be seen that the heat losses are maximal for  $n = 100$  rpm. This is because the voltage and so also the current decrease at decreasing rotational speed. In figure 6 it can be seen that the torque for a resistance load decreases linear to the decrease of the rotational speed. A PM-generator can also have high torques at low rotational speeds so the chosen value of the resistance gives only a large torque for the rated rotational speed  $n = 100$  rpm. To know the real capacity of the generator, it should also be measured for lower values of the resistance at lower rotational speeds than 100 rpm up to at least the rated torque  $Q = 1123.5$  Nm which is valid for  $n = 100$  rpm. As the current  $I$  is proportional to the torque  $Q$ , the copper losses and so  $P_{\text{heat}}$ , will then be the same for lower rotational speeds and the rated torque. The efficiency will be lower than for the given resistance load but this is acceptable.

A constant rated torque means that the power increases linear to the rotational speed. So a linear  $P$ - $n$  curve “rated torque” through  $P = 0$  W and  $n = 0$  rpm and  $P = 11765$  W and  $n = 100$  rpm is also drawn in figure 4. The line is extended to the right up to a power of 14000 W. Use of the generator below this line is certainly acceptable without getting a too high value of  $P_{\text{heat}}$ .

In figure 4 it can be seen that this line is intersecting with the optimum cubic line at a power of about 10800 W, belonging to a wind speed of about 8.1 m/s and a rotational speed of about 92 rpm. At a wind speed of 10 m/s, the optimum cubic line is lying somewhat left from the line for the rated torque and so the torque is somewhat larger than the rated torque. But the generator can supply a maximum torque which is even a lot higher than the rated torque of 1123.5 Nm and wind speeds above  $V = 8.1$  m/s will happen only during a small part of the time. So it is assumed that the pendulum safety system with a design wind speed of 8 m/s and so with a rated wind speed of 10 m/s, is a good choice and that the maximum power is limited enough for this design wind speed.

The generator efficiency is assumed to be 0.85 for a resistance load. If the optimum cubic line is followed, the real load is lower than the given resistance load if  $V$  is lower than 7.4 m/s. This means that the efficiency will be somewhat higher than 0.85. The real electrical power depends also on the losses in the rectifier and on the efficiency of the inverter. Rectifier losses are low for high voltages. Modern inverters have a very high efficiency. It is assumed that the total efficiency of generator, rectifier and inverter  $\eta_{\text{tot}} = 0.8$ .

The  $P_{el}$ - $n$  curve for an inverter load such that the optimum cubic line is followed and for a constant efficiency of 0.8 is also given in figure 4. The  $P_{el}$ - $n$  curve is determined for wind speeds up to 10 m/s. The working point for a certain wind speed is the point of intersection of the  $P$ - $n$  curve of the rotor for that wind speed with the optimum cubic line. The electrical power for a certain wind speed is found by going down vertically from the working point until the  $P_{el}$ - $n$  curve is crossed. The values of  $P_{el}$  have been determined this way for wind speeds up to 10 m/s and are given in the  $P_{el}$ - $V$  curve of figure 7. It is assumed that  $P_{el}$  for higher wind speeds than 10 m/s is the same as the value for  $V = 10$  m/s.

The electrical power is about 10.8 kW at a wind speeds of 10 m/s and higher which is very good for a windmill with a rotor diameter of 10 m and a design wind speed of 8 m/s. The  $P_{el}$ - $V$  curve is a lot better than that of the VIRYA-9 because of the larger rotor diameter and because the rotor is turning out of the wind only for wind speeds higher than 8 m/s.

If the generated energy is used to power a heat pump, about four times more heat is generated than the input electrical power. So even at moderate wind speeds, a substantial amount of heat will be generated by the VIRYA-10.

It is expected that the inverter needs a minimum input voltage to function. So the rotor must have a certain minimal rotational speed. This speed isn't known but at the moment it is supposed that the voltage is too low for wind speeds below 3 m/s. This means that the little energy available in wind speeds below 3 m/s can't be captured. So this is the reason why the  $P_{el}$ - $V$  curve starts suddenly with  $P_{el} = 448$  W at  $V = 3$  m/s. The critical voltage may lie lower and if this is the case, the  $P_{el}$ - $V$  curve starts at a lower wind speed. In chapter 4 it was calculated that the starting wind speed is only 1.5 m/s which is much lower than 3 m/s. So the rotor will turn almost always but it will generate no power for  $1.5 < V < 3$  m/s.

The  $P_{el}$ - $V$  curve is valid for constant wind speeds and not for average wind speeds. The output for a certain average wind speed is larger than for a certain constant wind speed. This can be demonstrated as follows. Assume we have a constant wind speed of 5 m/s. In the  $P_{el}$ - $V$  curve it can be read that  $P_{el} = 2074$  W. Assume we have a wind speed of 7 m/s for one hour and of 3 m/s for one hour. So the average wind speed is 5 m/s. The power for  $V = 3$  m/s is 448 W. The power for  $V = 7$  m/s is 5690 W. The average power is  $(448 + 5690) / 2 = 3069$  W. This is 995 W more or a factor  $3069 / 2074 = 1.48$  higher than for a constant wind speed of 5 m/s.

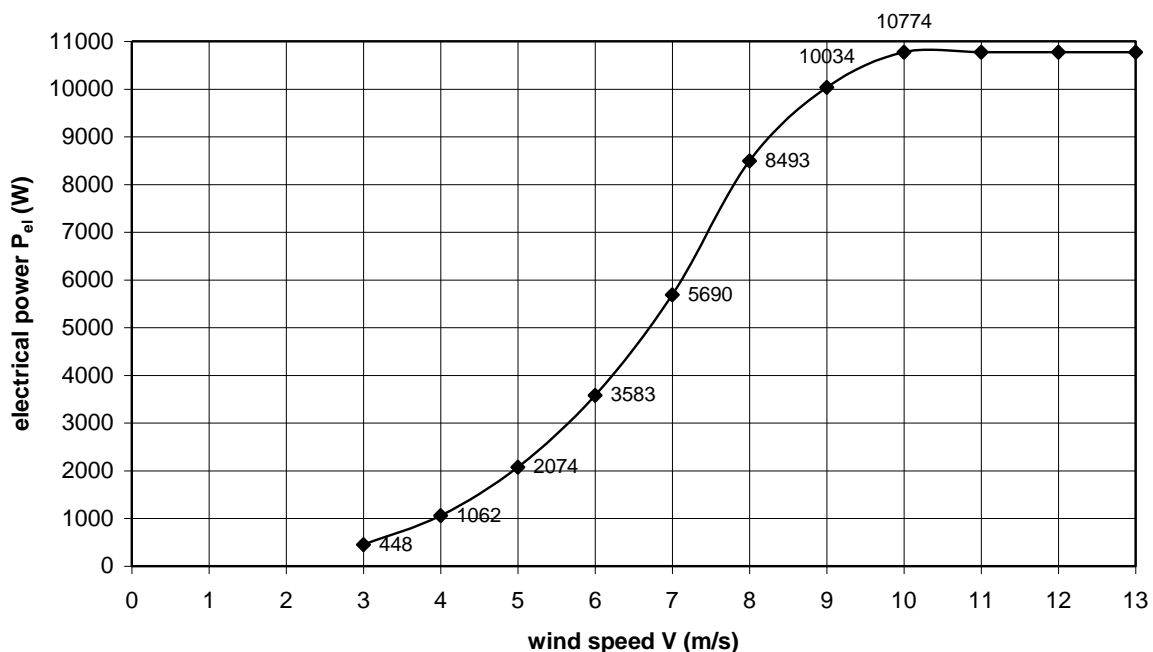


fig. 7  $P_{el}$ - $V$  curve for an inverter load such that the optimum cubic line is followed for  $3 \text{ m/s} < V < 10 \text{ m/s}$

The  $P_{\text{mech-n}}$ , the  $P_{\text{el-n}}$  curves as given in figure 4 and the  $P_{\text{el-V}}$  as given in figure 7 are estimated and not measured. Measured characteristics are more accurate than estimated characteristics. So to be sure that an acceptable matching is realised for the chosen generator of Hefei Top Grand, it is necessary to buy one and to test it at a large test rig with which it is possible to also measure the torque  $Q$ . One should also select and buy an inverter and measure the real electrical output for grid connection. Finally a complete windmill has to be built and tested with the correct load.

It might be possible to use the rotor as a brake to stop the rotor. To verify if this is possible at any wind speed, one should know the  $P$ - $n$  curve for short-circuit in star but this curve isn't given. The PM-generator which is used for the measurements as given in KD 78 (ref. 10) was measured for different constant voltages rectified in star. The  $Q$ - $n$  curves for 26 V star, 52 V star and 76 V star are given in figure 8 of chapter 4 of KD 78. The  $Q$ - $n$  curve for short-circuit in star before the rectifier is given in figure 4 of chapter 3 of KD 78. If these curves are compared, it can be seen that all curves have about the same shape but that the curve is shifted to the right if the voltage is higher. All curves have about the same maximum value of about 29 Nm. The first part of each curve, up to about 2/3 of the peak value, so up to a torque of about 20 Nm, is about a straight line but the curves bend to the right for higher torques. The curves start at the rotational speed for which the open generator voltage is equal to the average charging voltage. This phenomenon is used to derive the wanted  $P$ - $n$  curve for short-circuit in star for the VIRYA-10 generator.

In figure 5 it can be seen that the loaded DC voltage  $U_{\text{DC}} = 513$  V DC at  $n = 100$  rpm. The unloaded open DC voltage  $U_{\text{open}} = 623$  V at  $n = 100$  rpm. As the  $U$ - $n$  curves are straight lines through the origin, it can be read that  $U_{\text{open}} = 513$  V DC at  $n = 82$  rpm. This is 18 rpm lower than  $n_{\text{rated}} = 100$  rpm. Next it is assumed that  $Q$ - $n$  line for a constant voltage of 513 V is a straight line in between the point  $Q = 0$  Nm and  $n = 82$  rpm and the point  $Q = 1123.5$  Nm and  $n = 100$  rpm. This curve is also given in figure 6.

Short-circuit means a constant voltage  $U = 0$  V. So the  $Q$ - $n$  curve for short-circuit in star is found by moving the  $Q$ - $n$  curve for 513 V DC that much to the left that it intersects with the origin. This means that it must go through the point  $Q = 0$  Nm and  $n = 0$  rpm and the point  $Q = 1123.5$  Nm and  $n = 18$  rpm. This curve is also given in figure 6. Formula 12 can be written as:

$$P_{\text{mech}} = Q * \pi * n / 30 \quad (\text{W}) \quad (13)$$

To determine the  $P$ - $n$  curve for short circuit in star, several points have to be chosen on the  $Q$ - $n$  curve for short-circuit in star. This was done for every 2 rpm.  $P$  is then calculated for every point using formula 13. The result of this procedure is given in table 4. It is assumed that the  $Q$ - $n$  curve is about straight for rotational speeds up to 28 rpm and the  $Q$ - $n$  and  $P$ - $n$  curves are extended up to this rotational speed. For higher rotational speeds than about 28 rpm, the  $Q$ - $n$  curve for short-circuit in star will bend to the right and will have a maximum value at a certain rotational speed. This part of the curve can only be determined by measuring. So the  $P$ - $n$  curve for short-circuit in star can't be determined for rotational speeds higher than about 28 rpm.

n (rpm)	Q (Nm)	P (W)
0	0	0
2	124.8	26
4	249.7	105
6	374.5	235
8	499.3	418
10	624.2	654
12	749.0	941
14	873.8	1281
16	998.7	1673
18	1123.5	2118
20	1248.3	2614
22	1373.2	3164
24	1498.0	3765
26	1622.8	4418
28	1747.7	5125

table 4 Calculated values of Q and P as a function of n for short-circuit in star

The P-n curve for short-circuit in star can now be derived from table 4 and is also given in figure 4. It can be seen that there is a large distance in between the P-n curve for short-circuit in star and the P-n curve of the rotor for  $V = 10$  m/s. The P-n curve for short-circuit in star couldn't be determined for higher rotational speeds than 28 rpm but by interpolation it can be concluded that the generator can very well be used as a brake to stop the rotor at any wind speed.

Building of a prototype of the VIRYA-10 with the chosen PM-generator of Hefei Top Grand is only possible if a composite drawing is made and if detailed drawings are available but I won't make them. So only companies with enough engineering and manufacturing capacity should start with the VIRYA-10. The VIRYA-10 is certainly not a windmill which can be built by an amateur.



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